Chemistry 163B Lecture 09

**Carnot Arithmetic** 

**Challenged Penmanship** 

Notes

see handout: Carnot Arithmetic



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- 1. Phenomenological statements (what is ALWAYS observed)
- 2. Ideal gas Carnot *[reversible]* cycle efficiency of heat  $\rightarrow$  work (Carnot cycle transfers heat only at T<sub>U</sub> and T<sub>L</sub>)
- Any cyclic engine operating between T<sub>U</sub> and T<sub>L</sub> must have an equal or lower efficiency than Carnot OR VIOLATE one of the phenomenological statements (observations)
- 4. Generalize Carnot to any reversible cycle (E&R fig 5.4)
- 5. Show that for this REVERSIBLE cycle

 $q_U + q_L \neq 0$  (dq inexact differential  $\oint dq \neq 0$ ) but

$$\frac{q_U}{T_U} + \frac{q_L}{T_L} = 0 \quad (something \text{ VERY VERY special about } \frac{\mathrm{d}q_{rev}}{T}; \quad \oint \frac{\mathrm{d}q_{rev}}{T} = !!!)$$

6. S, entropy and spontaneous changes

- Disorder, **W**, did not change during an adiabatic reversible expansion  $(q_{rev} = 0)$
- Disorder, **W**, increased in isothermal reversible expansion  $(q_{rev} > 0)$
- Disorder, W, increased with T increase (q>0)
- Disorder, **W**, decreased with T decrease (q<0)
- As  $T \rightarrow 0$ ,  $W \rightarrow 1$

statements of the Second Law of Thermodynamics (roadmap #1)

- Macroscopic properties of an <u>isolated system</u> eventually assume constant values (e.g. pressure in two bulbs of gas\_becomes constant; two block of metal reach same T) [*Andrews.* p37]
- 2. It is impossible to construct a device that operates in cycles and that converts heat into work without producing some other change in the surroundings. *Kelvin's Statement [Raff p 157]; Carnot Cycle*
- 3. It is impossible to have a natural process which produces no other effect than absorption of heat from a colder body and discharge of heat to a warmer body. *Clausius's Statement, refrigerator*
- 4. In the neighborhood of any prescribed initial state there are states which cannot be reached by any adiabatic process
  - ~ Caratheodory's statement [Andrews p. 58]





### goals of Carnot arithmetic (step 2 of roadmap)

- 1. Carnot cycle is "engine" that produces work from heat
- net work for cycle: w<sub>surr</sub> >0 engine DOES work on surroundings
- Define efficiency: efficiency=(net work done by machine)/(heat energy input to machine)
- 3. Today, arithmetic manipulations of 1<sup>st</sup> Law results from ideal gas Carnot cycle (HW2 #10) to show that this efficiency depends only on the two temperatures at which heat is transferred to and from surroundings (the T<sub>U</sub> of step 1 and T<sub>L</sub> of step 3; the non-adiabatic paths)
- 4. Although for [reversible] Carnot cycle  $\oint dq_{rev} \stackrel{\text{WILL}}{\neq} 0$  but  $\oint \frac{dq_{rev}}{T} \stackrel{\text{WILL}}{\equiv} 0$



for system in complete cycle:

∆U=0; q >0; w <0 (work DONE on surr) (Prob #10e)

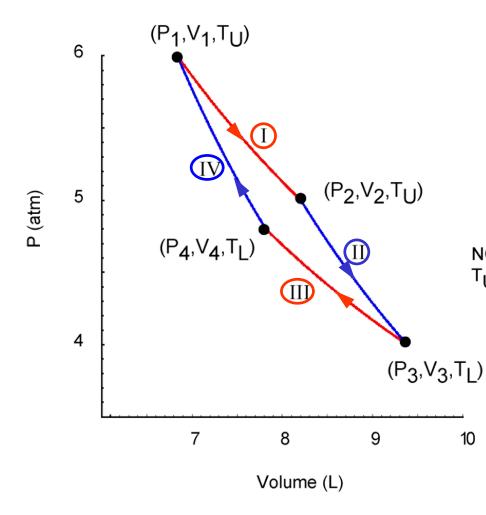
q > 0 ( $q_{in}$ ) at higher  $T_H$ ; q < 0 ( $q_{out}$ ) at lower  $T_L$ 

efficiency=  $-w/q_{1\rightarrow 2}$ (how much total [net] **work out** (-sign) for **heat in**  $1\rightarrow 2$ )

efficiency will depend on  $T_{\rm U}$  and  $T_{\rm L}$ 

HW4 prob #22 
$$\varepsilon$$
 is  $\varepsilon$ fficiency  
 $\varepsilon = \frac{T_H - T_C}{T_H}$  or  $\varepsilon = \frac{T_U - T_L}{T_U}$   
H=HOT C=COLD or U=UPPER L=LOWER

Problem HW2 #10 (see handout "Carnot Cycle Arithmetic)



*in prob #10*  

$$P_1=6 \text{ atm}, T_1=T_U=500K$$
  
 $P_2=5 \text{ atm}, T_2=T_U=500K$   
 $P_3=4 \text{ atm}, q_{2\rightarrow 3}=0, T_3=T_L$   
 $P_4=4.8 \text{ atm}, T_4=T_L, q_{4\rightarrow 1}=0$ 

NOTE:  $T_U (T upper) \equiv T_H (T higher or T hotter)$ 

### cyclic process

I isothermal expansion II adiabatic expansion III isothermal compression IV adiabatic compression • get  $w_I + w_{II} + w_{III} + w_{IV} = w_{total}$ 

• get  $q_I = q_{input}$ 

#### Summary (see handout "Summary of Heat and Work for the Carnot Cycle Engines, Refrigerators, Heat Pumps")

# general expressions for $(P_1, T_U) \xrightarrow{I} (P_2, T_U) \xrightarrow{II} (P_3, T_L) \xrightarrow{III} (P_4, T_L) \xrightarrow{IV} (P_1, T_U)$

ENGINE	q	W <sub>sys</sub>	W <sub>surr</sub>	
I. isothermal expansion	$+ nR T_v \ln \frac{P_1}{P_2}$ 1.3	$-nRT_U \ln \frac{P_1}{P_2} \qquad 1.2$	$+ nR T_{_U} \ln \frac{P_1}{P_2}$	heat in at T <sub>H</sub> work out
II adiabatic expansion	0	$n\overline{C_{_V}}(T_{_L}-T_{_U})$ 2.4	$-n\overline{C_{_V}}(T_{_L}-T_{_U})$	work out
III. isothermal	$nRT_L \ln \frac{P_s}{R} =$	$-nRT_L \ln \frac{P_s}{R}$	$-nRT_L \ln \frac{P_1}{P_2}$	heat lost at $T_L$
compression	$P_4$ 3.3&T.3 $-nR T_L \ln \frac{P_1}{P_2}$	$-nR T_{L} \ln \frac{P_{3}}{P_{4}}$ $= nR T_{L} \ln \frac{P_{1}}{P_{2}}$ 3.2&T.3	P <sub>2</sub>	work in
IV. adiabatic compression	0	$n\overline{C_v}(T_v - T_L)$ 4.4	$-n\overline{C_{_V}}(T_{_U}-T_{_L})$	work in
net gain/cost	$q_{in} = q_I$		w <sub>total</sub> = w <sub>l</sub> +w <sub>ll</sub> +w <sub>lll</sub> +w <sub>lV</sub> =	ε=w <sub>surr</sub> /q <sub>in</sub>
	$+ nR T_v \ln \frac{P_1}{P_2}$		$nR(T_{_U} - T_{_L})\ln\frac{P_{_1}}{P_{_2}}$	$\epsilon = (T_U - T_L)/T_U$

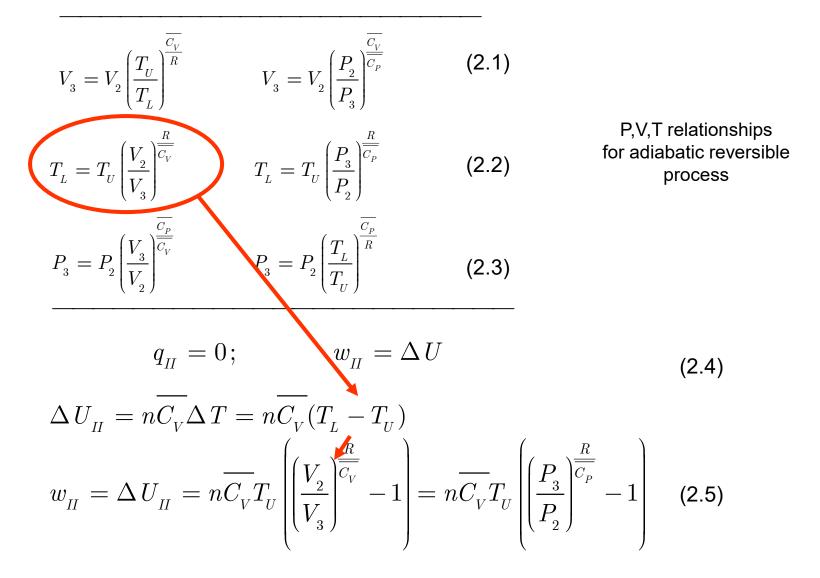
**Step I** Isothermal expansion,  $T_U, V_1 \rightarrow V_{2^1}$ 

$$\Delta U_I = 0 \tag{1.1}$$

$$w_{I} = -nRT_{U}\ln\frac{V_{2}}{V_{1}} = nRT_{U}\ln\frac{P_{2}}{P_{1}}$$
(1.2)

$$q_{I} = -w_{I} = nRT_{U} \ln \frac{V_{2}}{V_{1}} = nRT_{U} \ln \frac{P_{1}}{P_{2}}$$
(1.3)

step II: adiabatic reversible compression  $T_U \rightarrow T_L$ 

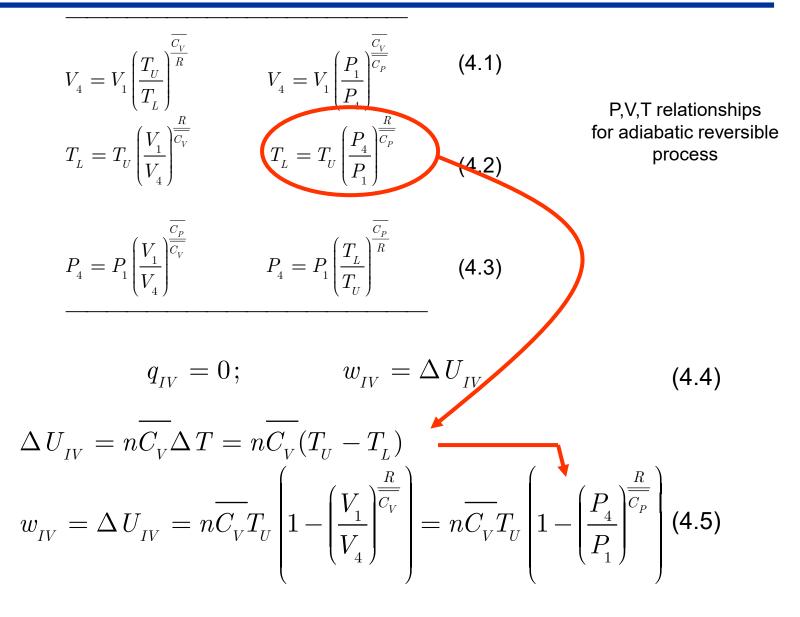


$$\Delta U_{III} = 0 \tag{3.1}$$

$$w_{III} = -nRT_L \ln \frac{V_4}{V_3} = nRT_L \ln \frac{P_4}{P_3}$$
(3.2)

$$q_{III} = -w_{III} = nRT_L \ln \frac{V_4}{V_3} = nRT_L \ln \frac{P_3}{P_4}$$
(3.3)

Step IV: adiabatic reversible compression  $(T_L \rightarrow T_U)$ 



**note:**  $\mathbf{w}_{IV} = -\mathbf{w}_{II}$  the two adiabatic steps have opposite work out  $\leftrightarrow$  work in

$$\Delta U_{II} = n \overline{C_V} (T_L - T_U) = w_{II} \quad \longleftarrow \quad w_{IV} = n \overline{C_V} (T_U - T_L) = \Delta U_{IV}$$

$$\begin{split} w_{II} &= n \overline{C_V} T_U \left( \left( \frac{P_3}{P_2} \right)^{\frac{R}{\overline{C_P}}} - 1 \right) = n \overline{C_V} T_U \left( \left( \frac{4}{5} \right)^{\frac{R}{\overline{C_P}}} - 1 \right) \\ & \bullet \\ w_{IV} &= n \overline{C_V} T_U \left( 1 - \left( \frac{P_4}{P_1} \right)^{\frac{R}{\overline{C_P}}} \right) = n \overline{C_V} T_U \left( 1 - \left( \frac{4.8}{6} \right)^{\frac{R}{\overline{C_P}}} \right) = n \overline{C_V} T_U \left( 1 - \left( .8 \right)^{\frac{R}{\overline{C_P}}} \right) \end{split}$$

and now for the TOTAL cycle ( $T_U$  and  $T_L$ ; and  $P_1$  and  $P_2$  given)

$$w_{\text{total}} = w_{\text{I}} + w_{\text{II}} + w_{\text{III}} + w_{\text{IV}}$$

$$w_{\text{II}} = -w_{\text{IV}} \Longrightarrow w_{\text{total}} = w_{\text{I}} + w_{\text{III}}$$

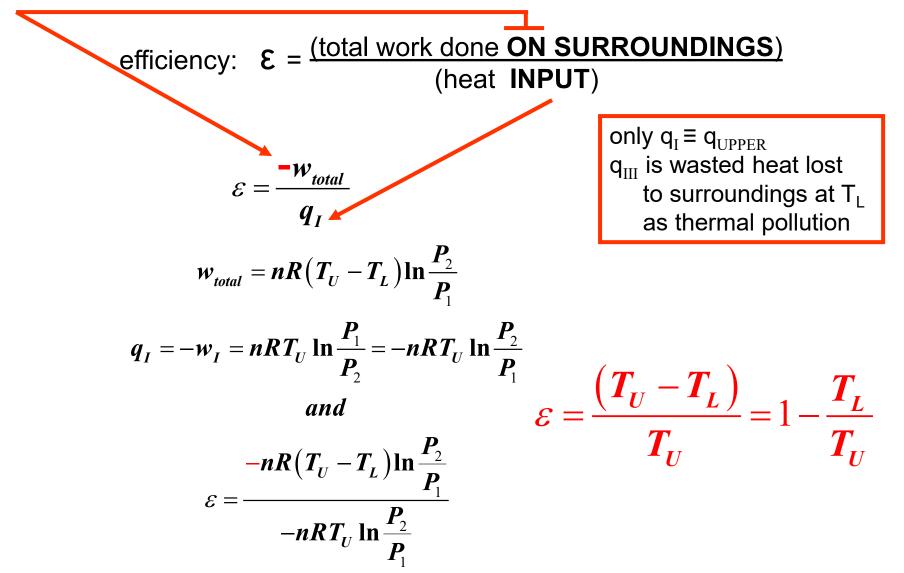
$$w_{I} = nRT_{U} \ln \frac{P_{2}}{P_{1}}$$

$$w_{III} = nRT_{L} \ln \frac{P_{4}}{P_{3}} \quad with \quad P_{3} = P_{2} \left(\frac{T_{L}}{T_{U}}\right)^{\frac{\bar{C}_{P}}{R}} \quad and \quad P_{4} = P_{1} \left(\frac{T_{L}}{T_{U}}\right)^{\frac{\bar{C}_{P}}{R}}$$

$$w_{III} = nRT_{L} \ln \left(\frac{P_{1} \left(\frac{T_{L}}{T_{U}}\right)^{\frac{\bar{C}_{P}}{R}}}{P_{2} \left(\frac{T_{L}}{T_{U}}\right)^{\frac{\bar{C}_{P}}{R}}}\right) = nRT_{L} \ln \left(\frac{P_{1}}{P_{2}}\right) = (-nRT_{L} \ln \left(\frac{P_{2}}{P_{1}}\right)^{-1})$$

$$w_{total} = nR(T_U - T_L) \ln \frac{P_2}{P_1}$$

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	$+ nR T_v \ln \frac{P_1}{P_2}$		$nR(T_{_U}-T_{_L})\ln\frac{P_{_1}}{P_{_2}}$	$\epsilon = (T_U - T_L)/T_U$

$$\varepsilon = \frac{\left(T_U - T_L\right)}{T_U} = 1 - \frac{T_L}{T_U}$$
  
lim  $\varepsilon = 0$ 

must have q in at higher and q out at lower T



$$\lim_{T_L \to 0} \mathcal{E} =$$

 $T_U \rightarrow \infty$ 

 $T_L \rightarrow T_U$ 

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perfect efficiency at finite temperatures only for T<sub>LOWER</sub> = 0 K



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6. S, entropy and spontaneous changes



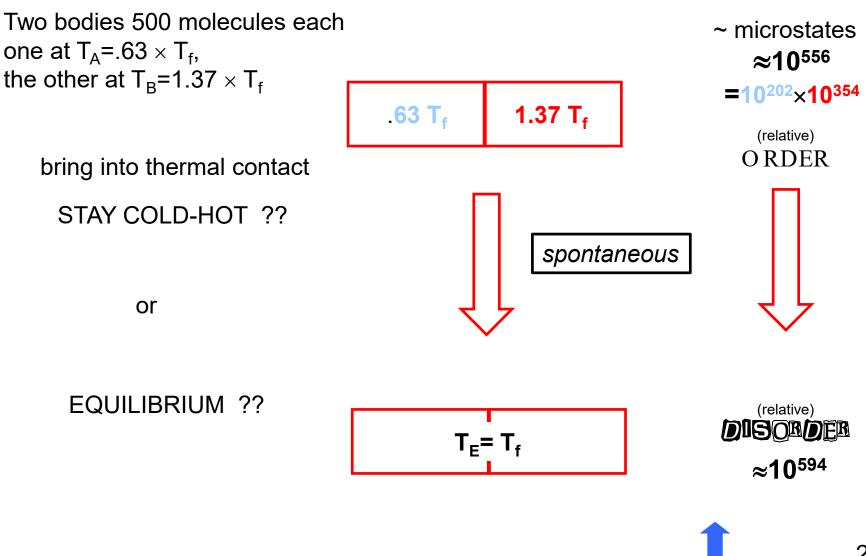


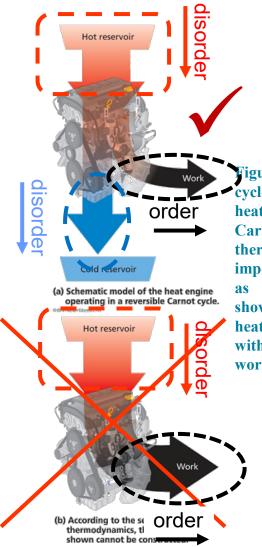
# End of Lecture 9





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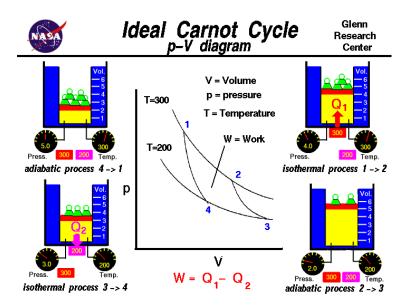


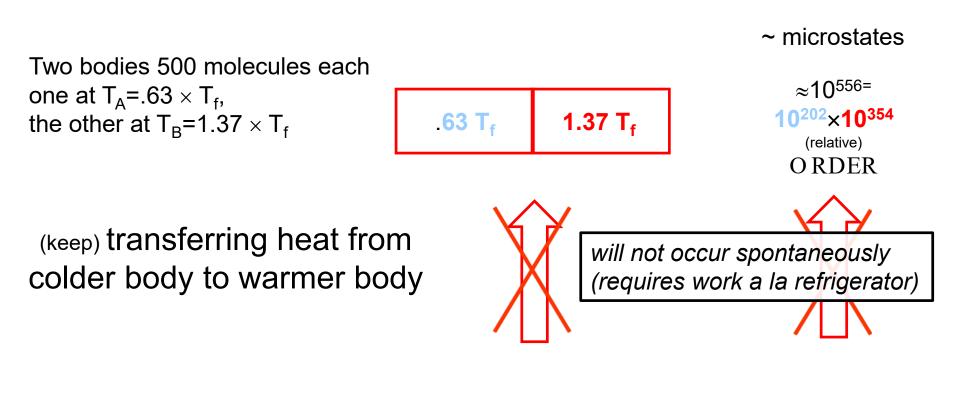
These considerations on the efficiency of reversible heat engines led to the Kelvin–Planck formulation of the **second law of thermodynamics:** 

It is impossible for a system to undergo a cyclic process whose sole effects are the flow of heat into the system from a heat reservoir and the performance of an equal amount of work by the system on the surroundings.

Figure 5.14 Carnot heat engine cycle. (a) A schematic model of a heat engine operating in a reversible Carnot cycle. (b) The second law of thermodynamics asserts that it is impossible to construct a heat engine

shown that operates using a single heat reservoir and converts the heat withdrawn from the reservoir into work with 100% efficiency as shown





Two bodies 500 molecules each at T<sub>f</sub>,

$$T_{E} = T_{f}$$
(relative)
(rela